



- ❖ Kinetic Energy Launcher: A projectile that has high kinetic energy due to its high speed and mass and is therefore highly efficient in creating a cavity in the target.
- ❖ Impact of the projectile (cross-section) diameter: The projectile diameter is one of the important and effective parameters in penetration. Increasing projectile diameter results in an increased cross-sectional area, and this factor reduces ballistic speed in ceramics that are brittle and exhibit brittle behavior. In other words, due to the ceramic brittle, the higher the cross-sectional area, the higher the surface of the ceramic being broken at the first moment of impact, thus facilitating projectile penetration.
- ❖ Mushrooming projectile shape: The flow of projectile head material after it hits the target in a radial direction and increases the projectile cross-section.
- ❖ Ricochet: The projectile crosses the surface of the striker without stopping or penetrating it.
- ❖ Fracture due to initial stress wave: occurs when the initial stress wave in an impacted zone exceeds the ultimate compressive strength σ_{uc} of the material. This can occur in weak and low-density targets.
- ❖ Radial fracture at the frontal side: occurs in the frontal side of the target. This is conceivable in brittle target elements whose tensile strengths are substantially lower than their corresponding compressive values, such as ceramics.
- ❖ Spalling and Scabbing is a tensile material failure due to the tensile reflection of the initial compressive transient waves from the distal side (far side) of the target and is a common phenomenon under explosive loading. Failure by spalling can occur on either the front or back of a target and is characterized by the formation of petals or ejects. Scabbing has a similar appearance, but the fracture is produced by deformation and its surface is determined by local inhomogeneity and/or anisotropies that may exist in the rolling direction.
- ❖ Petaling: is produced by high radial and circumferential tensile stresses after the passage of the initial wave occurring near the tip of the penetrator. This deformation is the result of bending moments created by the forward motion of the plate material being pushed ahead of the striker, and by inhomogeneity or planes of the weakness of the target. It is most frequently observed in thin plates struck by ogive or conical bullets at relatively low impact velocities or by blunt projectiles near the ballistic limit.
- ❖ Fragmentation: occur when the projectile strikes at high velocities on the brittle targets like ceramics and targets made of heterogeneous materials like concrete.
- ❖ Ductile failure or the ductile hole enlargement: the impact impulse overcomes the peripheral dynamic shear strength of the target material, pushing it outward and toward the impact surface to form a crater that is much larger than the projectile diameter. At the same time, the projectile pushes into the target, and there are hydrodynamic erosion and inversion of the penetrator material against the preceding face of the target.
- ❖ Plugging: develops as the result of a nearly cylindrical slug of approximately the same diameter as the bullet being set in motion by the projectile. Failure occurs due to large shears produced around the moving slug. The heat generated by the shear deformation is restricted to a narrow annulus in which it decreases the material strength resulting in instability and is called an adiabatic shearing process. Plugging is most frequently found when blunt penetrators strike intermediate or thin, hard target plates. Its presence is sensitive to velocity and the angle of obliquity of pointed projectiles. Shear plugging is generally observed for thick targets, particularly with high strength materials. In these instances, an intense shear band may be observed intersecting a tensile opening at the stretched rear surface.
- ❖ Discing failure: in the case of discing, shear cracks develop in the plane of the plate as a consequence of in-plane shear stresses induced by bending. Both metallurgical inclusions and inhomogeneity in the plane



of the plate, as well as adiabatic thermal softening effects associated with the high rate of deformation, contribute to discing failure.

- ❖ Dishing: occurs in targets of thinner plates where bending is favored. The stretching of the sheet can lead to tensile failures at the edges with a plug ejected, or the plug folded away attached to one of the petals, or necking and tearing in the form of a star pattern from the center of impact. This last mechanism involves bending and ironing the flat of the petals as well as radial stretching.

Fig. 2 shows different failure modes in impacted plates and Fig. 3 shows the effect of the impact below the ballistic limit on the projectile.

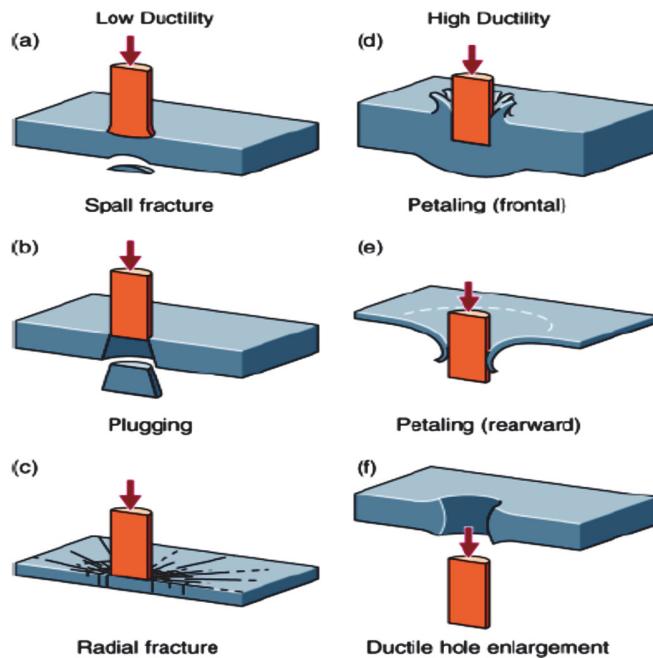


Figure 2: Different failure modes in impacted plates [10].

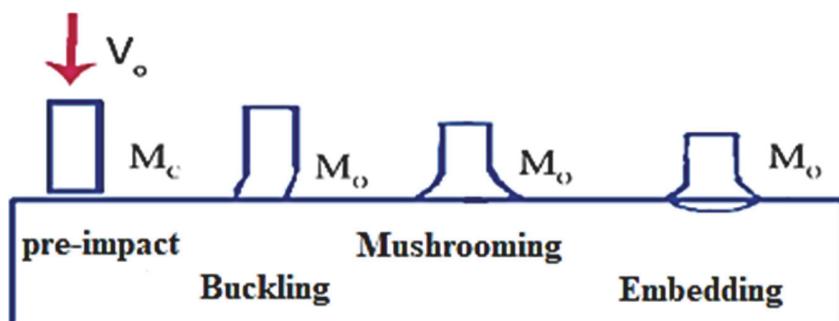


Figure 3: The effect of the impact below the ballistic limit on the projectile [10].

EFFECTIVE PROJECTILE AND TARGET PARAMETERS IN THE PENETRATION PROCESS

Parameters such as material and mechanical properties, nose shape, projectile diameter, velocity, mass, and angle of impact are important parameters of the projectile penetration process, which is the most important parameter in determining the type of failure [17-18].

- 1) Mechanical Properties and Material of the Projectile: One of the properties considered in ballistic penetration is the mechanical and material properties of the projectile. The projectile can generally be classified as follows.



Tab. 4 Mechanical Properties of Armored Ceramics and Tab. 5 The amount of fracture toughness in different materials and percentage Contribution of research on reinforcements and ceramics is shown in Fig. 7.

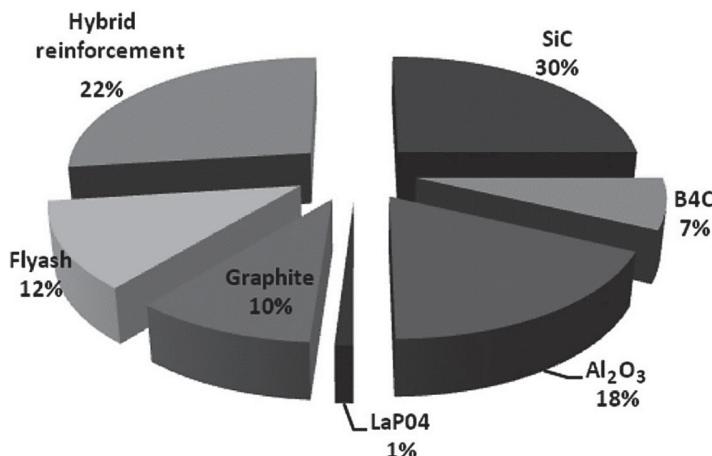


Figure 7: % Contribution of research on reinforcements and ceramics [21].

ANALYTICAL MODELS OF PENETRATION ON CERAMICS

This section examines the theories and models presented in the field of ceramics penetration.

T

Model by Johnson Cook

The relationship proposed by Johnson and Cook to express the effects of plastic work, plastic strain rate, and temperature on yield stress is given by the Eqn. (1):

$$\sigma = [A + B\varepsilon^n][1 + C \ln \dot{\varepsilon}^*][1 - T^{*m}] \quad (1)$$

where A, B, C, n, m the constants of the material and ε the strain of the plastic equivalent $\dot{\varepsilon}^*$ are the dimensionless parameters of the strain rate of the plastic $\dot{\varepsilon}^* = \dot{\varepsilon} / 1.0 s^{-1}$ to be defined. T^{*m} The dimensionless parameter is the temperature, which is calculated from the Eqn. (2).

$$T^{*m} = \frac{T - T_{Room}}{T_{Melt} - T_{Room}} \quad (2)$$

In this model, the effects of plastic strain rate overtime on yield stress are considered, but in Steinberg's model, it is ignored. The reason is the difference in the range of use of these models. In the experiments performed by Johnson and Cook to calculate the coefficients used in this model, the highest plastic strain rate was $400 s^{-1}$ but in the Steinberg experiments, they were more than $10^5 s^{-1}$ because of their attention to explosive loading and the extremely high velocity impacts. The equivalent plastic strain is obtained from Eqn. (3):

$$\varepsilon = \sum \dot{\varepsilon} \Delta t \quad (3)$$

The rate of change of the work done by the beams on the joint surface of the ceramic backing layer as shown in Fig. (9) is:

$$\frac{d\Gamma}{dt} = \pi f_b R^2 c w \quad (17)$$

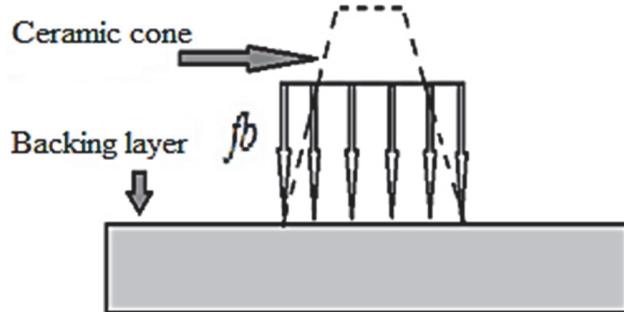


Figure 9: Ceramic cone and force applied to the backing layer [10].

where W is the surface velocity of the ceramic backing joint. Therefore, the kinetic energy conversion rate for the active region of the backing layer is:

$$\frac{dE_k}{dt} = \pi R^2 h_b \rho_b w \frac{dw}{dt} \quad (18)$$

Given the energy balance:

$$\frac{d\Gamma}{dt} = \frac{dE_p}{dt} - \frac{dE_k}{dt} \quad (19)$$

$$f_b R^2 c = h_b Y_b \left(\frac{2}{3} h_b + \delta \right) + R^2 h_b \rho_b \frac{dw}{dt} \quad (20)$$

From the above equation, we can calculate W . At very high impact velocities, the conical ceramics may be completely eroded, and therefore the projectile may come in contact with the back material. In this case, the actual behavior represents the speed difference between the projectile and the metallic material. Based on the Tate equations, the projectile motion equation can be modeled as follows:

$$\frac{dv}{dt} = - \frac{Y_b + \frac{1}{2} \rho_b (v - w)^2}{\rho_b L} \quad (21)$$

The equation of motion of the back material is given by the following Eqn. (17):

$$\frac{dw}{dt} = \frac{\frac{1}{4} \pi Y_b \frac{D_{eq}^2}{4} - \pi h_b Y_b \left(\frac{2}{3} h_b + \delta \right)}{M_b} \quad (22)$$

where M_b is the effective mass of the area and is equal to:



$$M_b = \pi \rho_b \left[R^2 b_b - \frac{D_{eq}^2}{4} (b_b - h_{bt}) \right] \quad (23)$$

where b_{bt} is the actual thickness of the center of the plate as shown in Fig. 10. Therefore, the analytical equations presented can calculate the projectile velocity and the velocity of the backing layer at any time interval. If the speed difference between them is zero, the projectile stops.

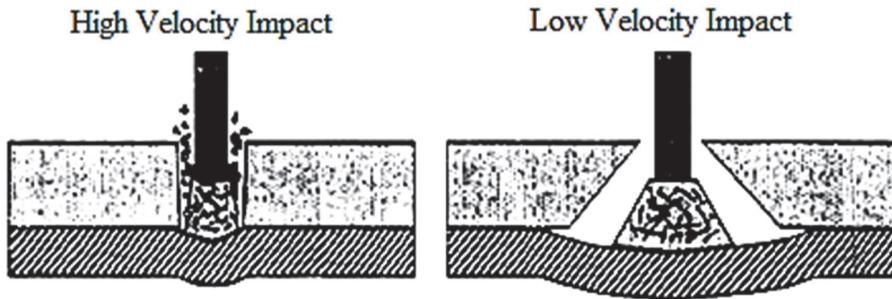


Figure 10: Full ceramic erosion and projectile contact with the back material [10].

There are two different yield criteria for defining full armor penetration in this model. The first criterion for cases where the speed is well above the speed of the ballistic limit. In this case, the projectile completely erodes the ceramic and exits the metal upon contact with the metal without causing any twist. In this case, the yield criterion is:

$$h_{bt} = 0 \quad (24)$$

The second criterion for states that are high enough to cause a significant twist in the metal (velocities close to the ballistic limit speed and slower than that). As can be seen in the numerical simulation, when the projectile speed is close to the metal velocity, failure occurs even when the full penetration of the ceramic or metal is not achieved. Therefore, in this case, a kinetic failure criterion is selected and the armor is assumed to fail when:

$$v = w \quad (25)$$

Model by Fellows

This model investigates the penetration of projectiles into thick armor with high velocities. The basis of this theory is the method of mass accumulation. Fig. (11) shows the schematic response of system mass accumulation. In this analysis, the joint surface between the projectile, the surface of the eroded projectile, the front ceramic surface, the surface of the eroded ceramic back, the surface of the eroded backing layer and the backing layer are considered [2].

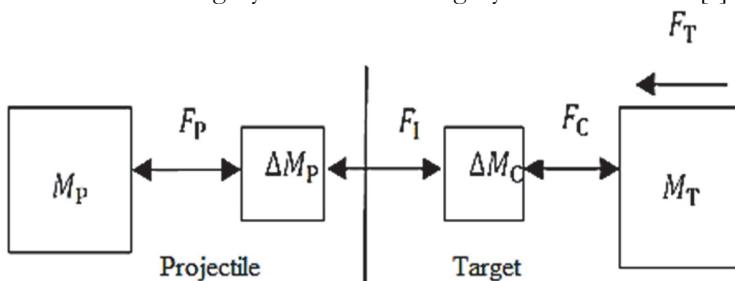


Figure 11: Reactions during a collision [10].

First, during high velocity impacts, the pressure at the joint surface of the ceramic projectile is greater than the erosive strength (a material required for erosion) of the projectile and thus the projectile is eroded. If the collision speed is high enough. The erosion strength of the ceramic will be excessive and the ceramic will also erode. As a result of projectile penetration in the ceramic, a conical crack is formed in the ceramic that transfers the load onto the backing layer. This is the case for ceramics. If the pressure at the ceramic joint surface of the backing layer is not high enough (less than the



strength of the backing material's erosion), it is assumed that the projectile penetrates the ceramic and forms a smaller new cone. At the ceramic surface of the backing layer more than the erosive strength of the backing material, the ceramic cone begins to penetrate the backing material. Using mass accumulation theory, the basic equations of this theory can be written as follows: Newton's second law of the application for projectiles:

$$F_p = -M_p \dot{X}_p \quad (26)$$

Newton's Second Law Application for Eroded Ceramic Front Panel:

$$F_1 - F_e = \frac{\Delta M_c \dot{X}_{CF}}{\Delta t} \quad (27)$$

Projectile Mass Reduction:

$$\Delta M_p = (\dot{X}_p - \ddot{X}_{CF}) \pi \rho_c \quad (28)$$

Reduce ceramic mass from the ceramic front panel

$$\Delta M_c = (\dot{X}_{CF} - \ddot{X}_c) \pi \rho_c \quad (29)$$

Projectile forces when the projectile is eroded or mushrooming is formed:

$$F_p = \sigma_{PES} \quad (30)$$

The relationship between ceramic front velocity, ceramic erosion, and ceramic velocity is:

$$\dot{X}_{CF} = \dot{X}_{CE} + \dot{X}_C \quad (31)$$

The joint surface force F_1 will be different for different phases of erosion, mushrooming shape, and projectile rigidity. In the above equations \dot{X}_p and \ddot{X}_p projectile velocity and acceleration at any instant, \dot{X}_{CF} ceramic front element speed, \dot{X}_C ceramic element velocity, and σ_{CES} and σ_{PES} the erosion stresses are ceramic and projectile.

Model by Forrestal and Luk

Forrestal and Luk proposed a model for penetrating a projectile with a spherical nose. To calculate the force applied to the projectile during penetration, they must consider the effect of frictional force in addition to normal stress. To do this, the tangential stress was defined as follows [12]:

$$\sigma_t = \mu \sigma_n \quad (32)$$

where μ is the slip friction coefficient between the target material and the projectile.

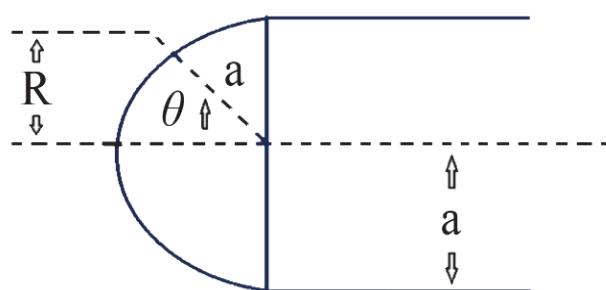


Figure 12: Cylindrical projectile with spherical nose [10].



According to Fig. (12), the forces resulting from normal and tangential stresses on the projectile nose can be obtained as follows:

$$dF_n = 2\pi Ra \sigma_n (V_x \cdot \theta) d\theta \quad (33)$$

$$dF_t = 2\pi Ra \mu \sigma_n (V_x \cdot \theta) d\theta \quad (34)$$

This axial force enters on a projectile penetrating V_x is equal to:

$$F_x = \pi a^2 \int_0^{\pi/2} \sigma_n (V_x \cdot \theta) [\sin 2\theta + 2\pi \sin^2 \theta] d\theta \quad (35)$$

Given this equation, the axial force applied to the projectile with the spherical nose can be calculated for the time it hits the target vertically. Also, the normal stress on the projectile nose, in Eqn. (30), can be approximated by the radial stress on the surface and the velocity of the target particles at the joint surface of the target nose will be affected by the rigid projectile penetration at the velocity V_x :

$$V(V_x \cdot \theta) = V_x \cos \theta \quad (36)$$

The depth of penetration of the projectile can then be calculated by obtaining the force applied to the projectile by Newton's law and the rigid equations of motion. The Forrestal model is one of the most widely used models of rigid penetration in targets and has been studied by many people. If the projectile nose shape is taken as a function of $y = y(x)$, the relationships are obtained as follows:

$$N_1 = 1 + \frac{8\mu}{d^2} \int_0^b y dx \quad (37-a)$$

$$N_2 = N^* + \frac{8\mu}{d^2} \int_0^b \frac{y'^2}{1+y'} dx \quad (37-b)$$

$$N^* = \frac{8}{d^2} \int_0^b \frac{y'^3}{1+y'} dx \quad (37-c)$$

where b is the length of the projectile's nose. Also, a dimensionless parameter for different projectile shapes is defined as follows:

$$\Psi = \frac{S}{d} \quad (38)$$

when the projectile is ogive, Ψ it is called the Caliber Radius Head. Thus, according to different projectile shapes and by using the Eqns. (37a) to (37c), the coefficients of shape can be obtained for different projectiles. This value for projectiles ogive in Fig. 13 is equivalent to:

$$N_1 = 1 + 4\mu\Psi^2 \left[\left(\frac{\pi}{2} - \varphi_0 \right) - \frac{\sin 2\varphi_0}{2} \right] \quad (39-a)$$

$$N_2 = \mu \Psi^2 \left[\left(\frac{\pi}{2} - \varphi_0 \right) - \frac{1}{3} \left(2 \sin 2\varphi_0 + \frac{\sin 4\varphi_0}{4} \right) \right] \quad (39-b)$$

$$N_2 = \frac{1}{3\Psi} - \frac{1}{24\Psi^2}; \quad 0 < N^* \leq \frac{1}{2} \quad (39-c)$$

$$\varphi_0 = \sin^{-1} \left(1 - \frac{1}{2\Psi} \right); \quad \Psi \geq \frac{1}{2} \quad (39-d)$$

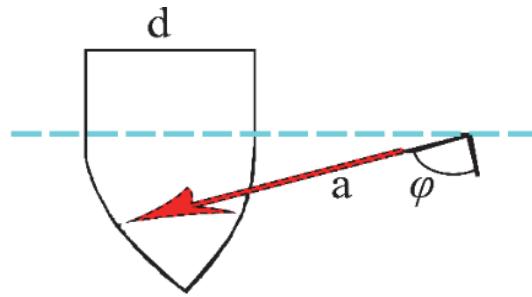


Figure 13: Projectile with ogive nose [10].

For flat nose projectiles in Fig. (14):

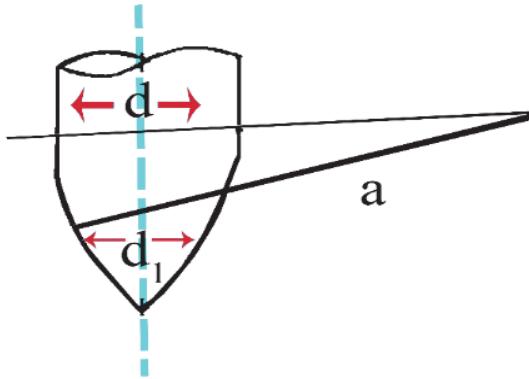


Figure 14: Ogive projectile with a flat nose [10].

$$N_1 = 1 + 4\mu \Psi^3 \left[\left(\frac{\pi}{2} - \varphi_0 \right) + \frac{\sin 2\varphi_0}{2} - 2 \left(1 - \frac{1}{2\Psi} \right) \cos \varphi_0 \right] \quad (40-a)$$

$$N_2 = \mu \Psi^3 \left[\left(\frac{\pi}{2} - \varphi_0 \right) + \frac{\sin 4\varphi_0}{4} - \frac{8}{3} \left(1 - \frac{1}{2\Psi} \right) \cos^3 \varphi_0 \right] \quad (40-b)$$

$$N_2 = \Psi^3 \left[2 \cos^4 \varphi_0 - \frac{8}{3} \left(1 - \frac{1}{2\Psi} \right) (2 + \sin \varphi_0) (1 - \sin \varphi_0)^2 \right] + \zeta^2; \quad \zeta = \frac{d_1}{d} \quad (40-c)$$

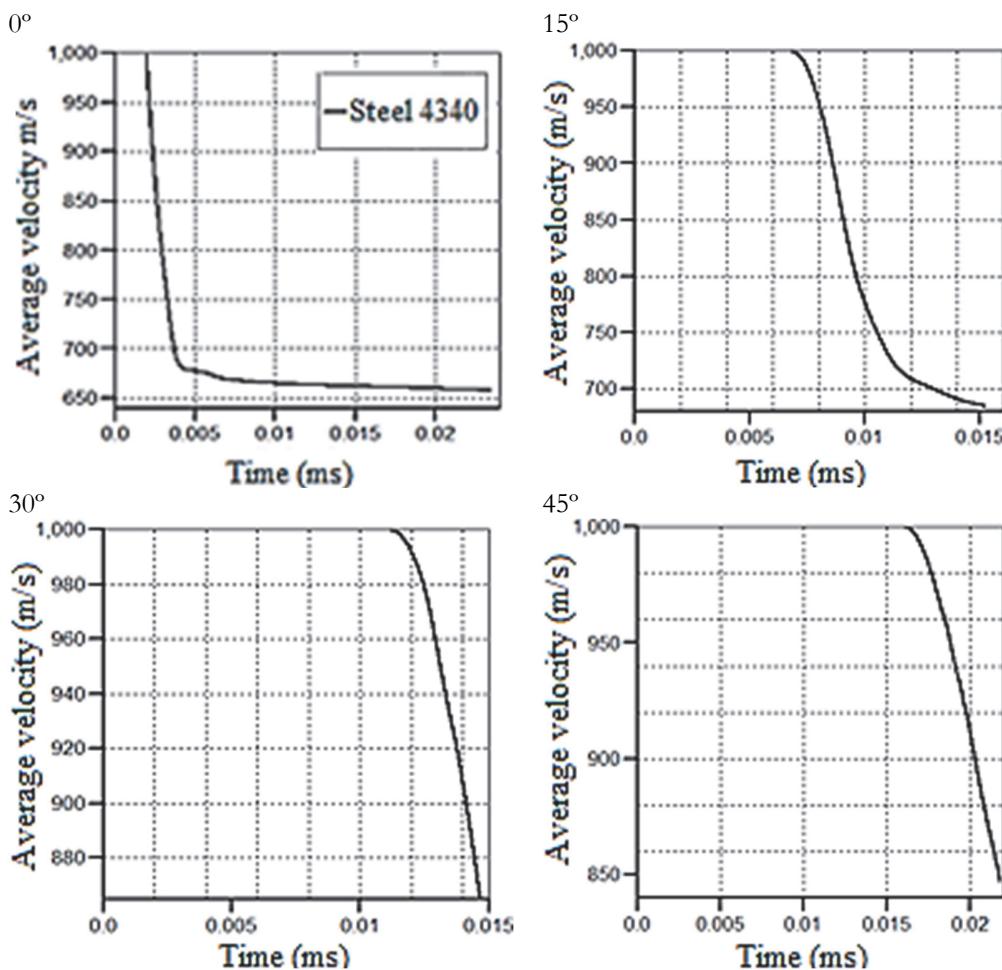


Figure 18: Projectile collision velocity to time at different angles.

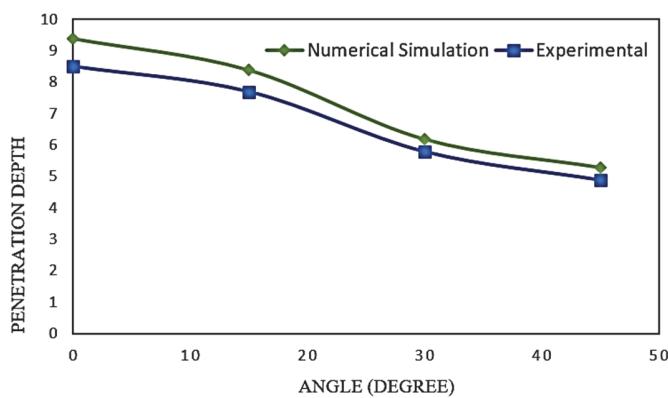


Figure 19: Comparison of the depth of penetration in numerical simulation and experimental data [10].

Tab. 7 shows the penetration depth of the ceramic target in each experiment and, as shown in the diagram in Fig. 19, as the angle of oblique decreases, the amount of penetration in the target decreases. Meshing and impact of the projectile on the ceramic target shown in Fig. 20.

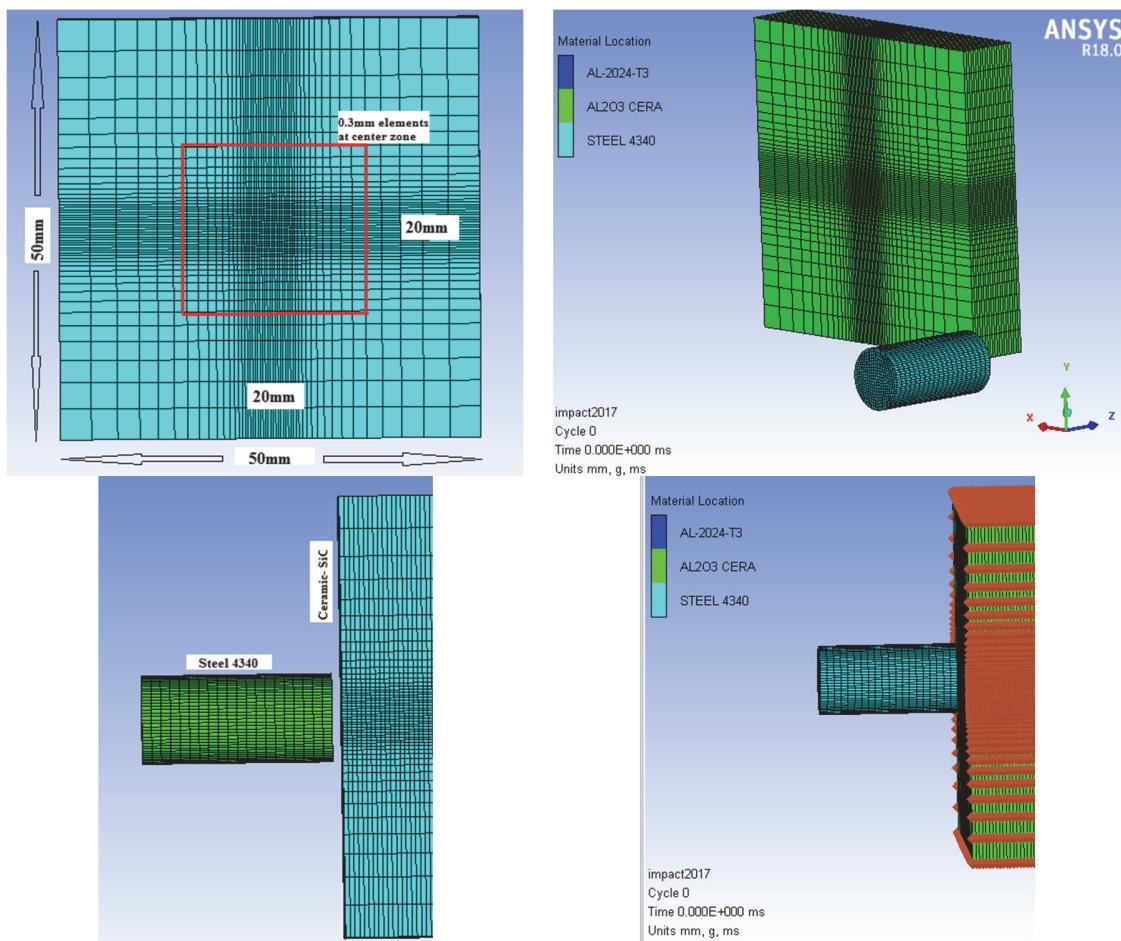


Figure 20: Meshing and the impact of the projectile on the ceramic target.

In contact between projectile and target, the most common type of contact is the definition of Contact Automatic Surface to Surface. Alongside this type of contact, there is also erosive contact. The problem with automatic contact for solid elements is that the projectile passes the target, but the elements that reach the ultimate stress and strain are not eliminated from the analysis. But with erosive contact, this problem can be resolved. In a projectile impact on a target, a three-dimensional contact is defined as a type of erosive contact, which is the reason for selecting this contact, penetrating the projectile into the target, and removing the elements. The kinetic energy of the projectile is reduced by removing its elements. Tab. 8 shows the Characteristics of Johnson Cook's Material Model (JH-1) for ceramic purposes, Tab. 9 shows the Characteristics of Johnson Cook's Material Model (JH-1) for steel projectiles.

CONCLUSIONS

Using the analytical equations and models presented in this paper as well as numerical simulation the following results are obtained.

1. By increasing the initial projectile velocity, the blunt projectile performance improves.
2. When a projectile hits a target, due to axial asymmetry, there is a torque on the plate that causes the projectile to deviate from its initial angle of impact. If the angle of impact of the projectile reaches the target, this torque can cause the projectile to have ricocheted off the target surface. All of these factors can lead to the complexity of analyzing and investigating oblique penetration processes.
3. The moment of impact, the Blunt projectile exhibits a higher surface area because of its brittle ceramics and fragile behavior, therefore, in the collision of the Blunt projectile, the surface is destroyed and the surface that is broken



- [23] Garcia, M. (1996). Ballistic Performance of Ceramic Faced Composite Plates (2nd Report), Journal of High-Pressure Institute of Japan, 34. DOI: 10.1016/j.mspro.2014.07.571.
- [24] Zaera, R. and Sánchez-Galvez, V. (1998). Analytical modeling of normal and oblique ballistic impact on ceramic/metal lightweight armors, International Journal of Impact Engineering, 21(3), pp. 133-148. DOI: 10.1016/S0734-743X(97)00035-3.
- [25] Chocron, S., Rodriguez, J. and Sanchez Galvez, A. (1997). A Simple Analytical Model to Simulate Textile Fabric Ballistic Impact. DOI: 10.1177/004051759706700707.
- [26] Awerbuch, J., Bodner, S. R. (1973). Analysis of the mechanics of perforation of projectiles in metallic plates, International Journal of Solids and Structures, 10, pp. 671-684. DOI: 10.1016/0020-7683(74)90050-X.
- [27] Ravid, M., Bodner, S. R. (1994). Penetration into thick targets refinement of a 2D dynamic plasticity approach, International Journal of Impact Engineering, 15, pp. 491-499. DOI: 10.1016/0734-743X(94)80030-D.